

Condotte in pressione

Ipotesi: moto permanente

Equazione di Bernoulli



$$\Delta H = L \cdot J + \sum P_c$$

Equazione di Darcy-Weisbach



$$J = \lambda \frac{V^2}{2gD} = \lambda \frac{Q^2}{2gDA^2}$$

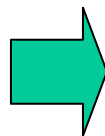
$$\frac{1}{\sqrt{\lambda}} = -2 \cdot \log_{10} \left(\frac{2.51}{\text{Re} \cdot \sqrt{\lambda}} + \frac{1}{3.71} \frac{\varepsilon}{D} \right)$$

Colebroke-White

$$\frac{1}{\sqrt{\lambda}} = -2 \cdot \log_{10} \left(\frac{5.8}{\text{Re}^{0.9}} + \frac{1}{3.71} \frac{\varepsilon}{D} \right)$$

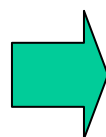
Cozzo

Equazione di Chezy (moto p. turbolento)



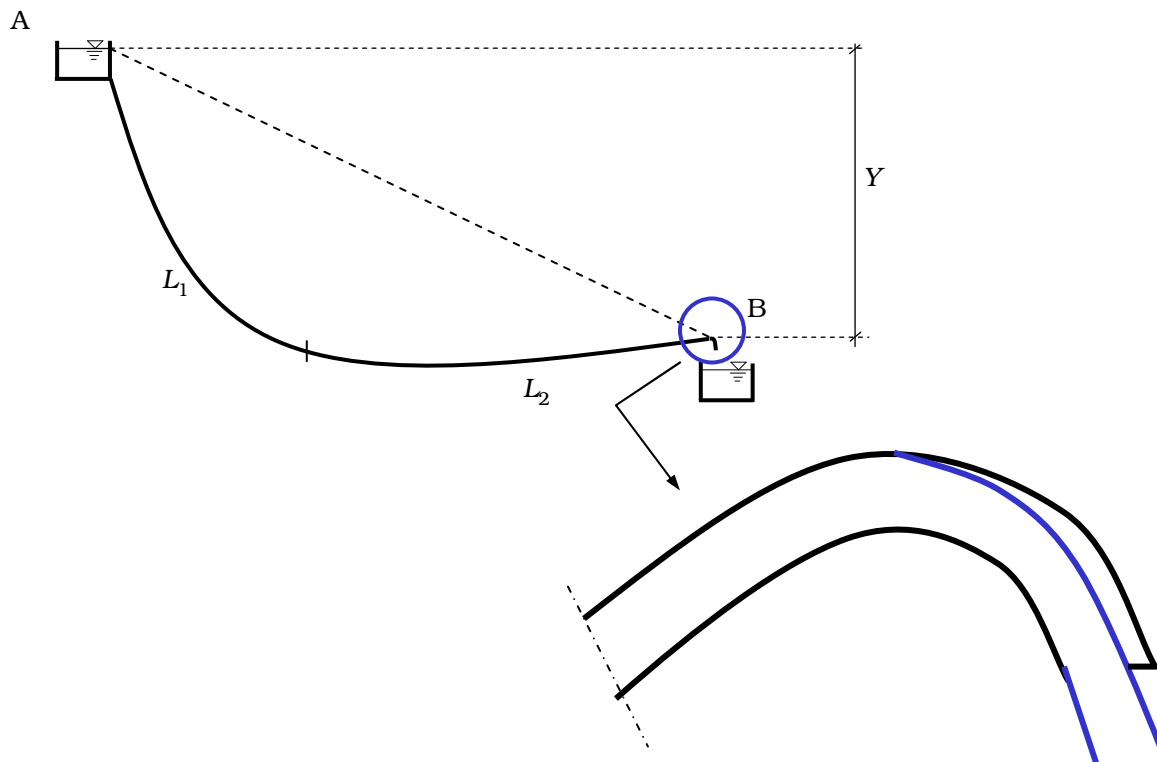
$$J = 4 \cdot \frac{V^2}{\chi^2 \cdot D}$$

Equazione di Chezy -Strickler

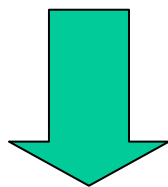


$$J = \frac{10.29}{K_s^2} \cdot \frac{Q^2}{D^{5.33}}$$

Condotta semplice: a) Verifica

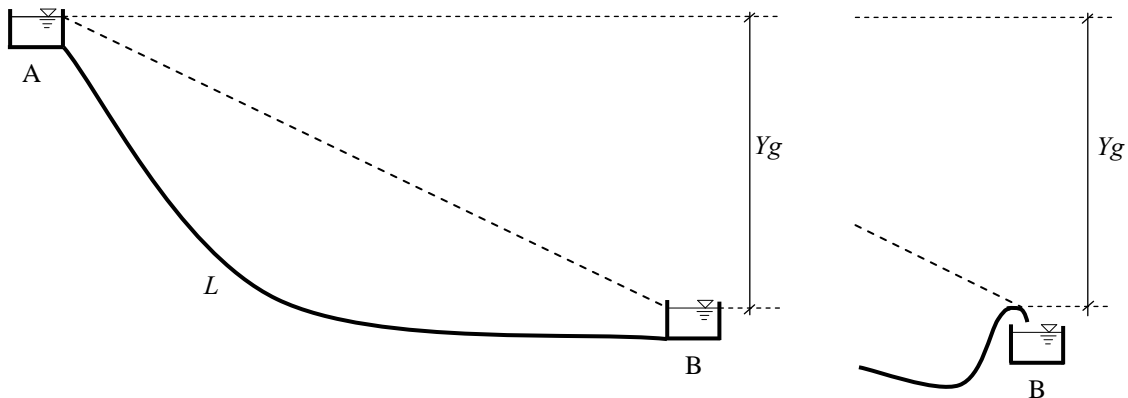


$$Y = z_A - z_B = L \cdot J = L \cdot \frac{10.29}{K_s^2} \cdot \frac{Q^2}{D^{5.33}}$$



Noti Y , D , L e K_s si ricava Q

Verifica di una condotta semplice sempre inferiore alla congiungente i due serbatoi

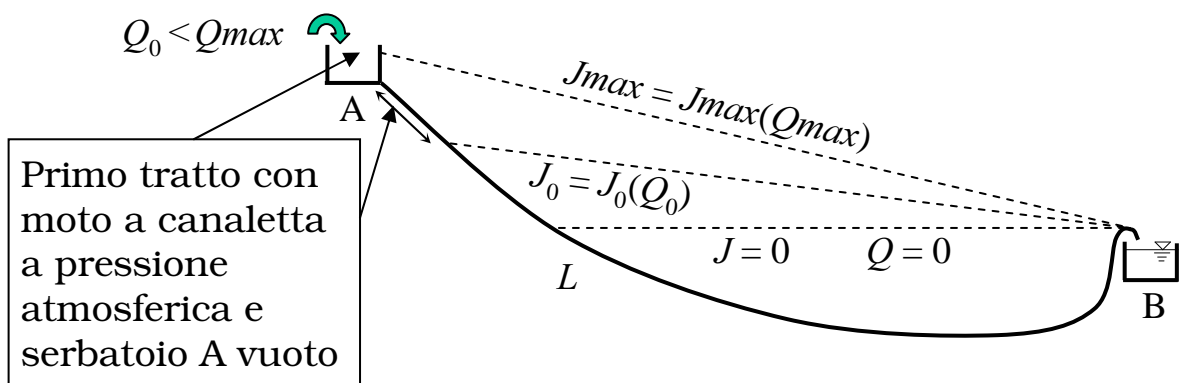


1) Calcolo della Q_{max} $Yg = L \cdot J \rightarrow Q_{max}$

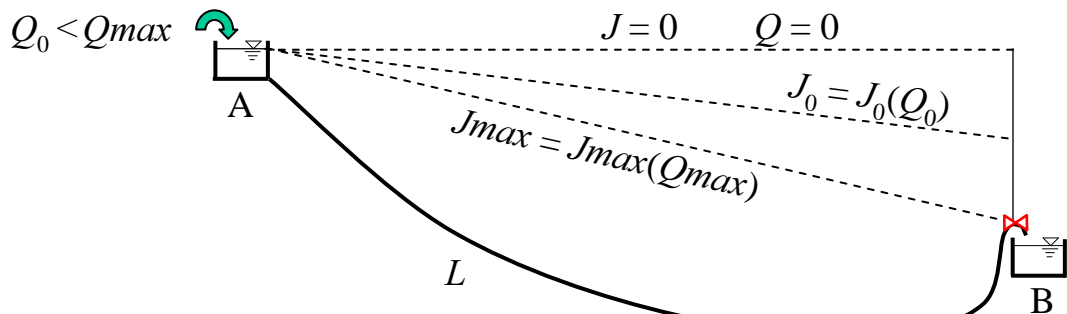
Condotte nuove e usate con piezom. e J identiche.

$J = K \cdot \frac{Q^\alpha}{D^m}$ Cambia la scabrezza (e quindi K), a volte pure il diametro D , e dunque la portata Q .

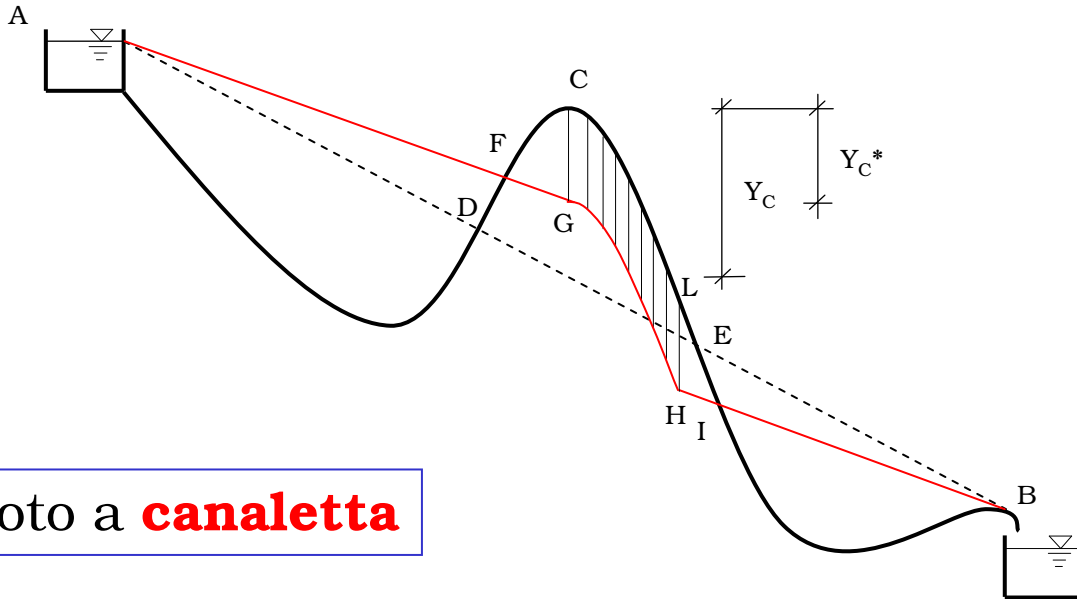
2) Calcolo piezometrica per una imposta $Q_0 < Q_{max}$



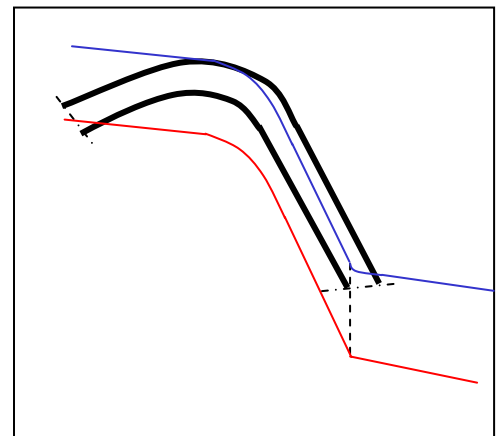
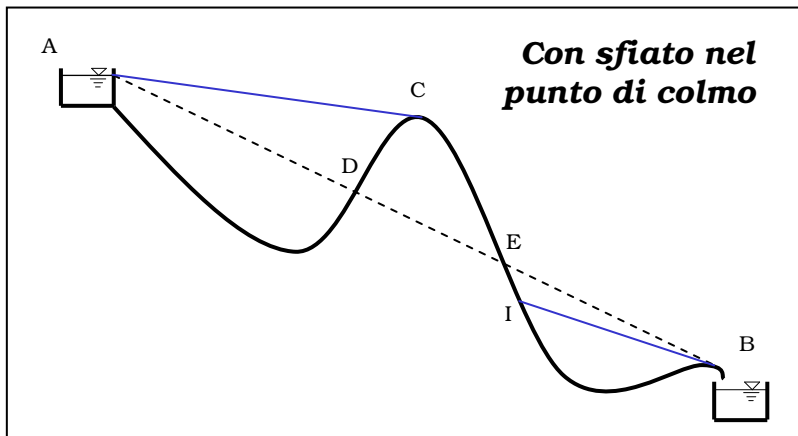
Per evitare ciò viene inserita una **valvola dissipatrice**:



Poi si verificano velocità e pressioni.



Moto a **canaletta**



Condizione limite:

$$\frac{P}{\gamma} > -10 m$$

(in teoria)

$$\frac{P}{\gamma} > -7 \div 8 m$$

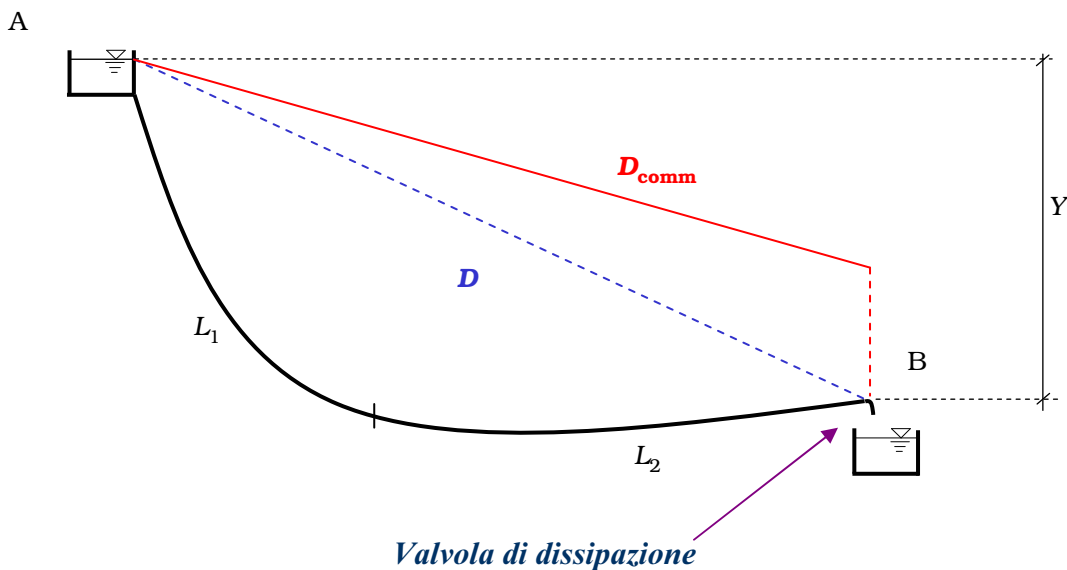
(in pratica)

Condotta semplice: b) progetto

Noti Y , Q e L e si ricava D

Si considera la condizione di tubi usati: $\rightarrow K_s$

$$Y = z_A - z_B = L \cdot J = L \cdot \frac{10.29}{K_s^2} \cdot \frac{Q^2}{D^{5.33}}$$



**Vincoli
ingegneristici**

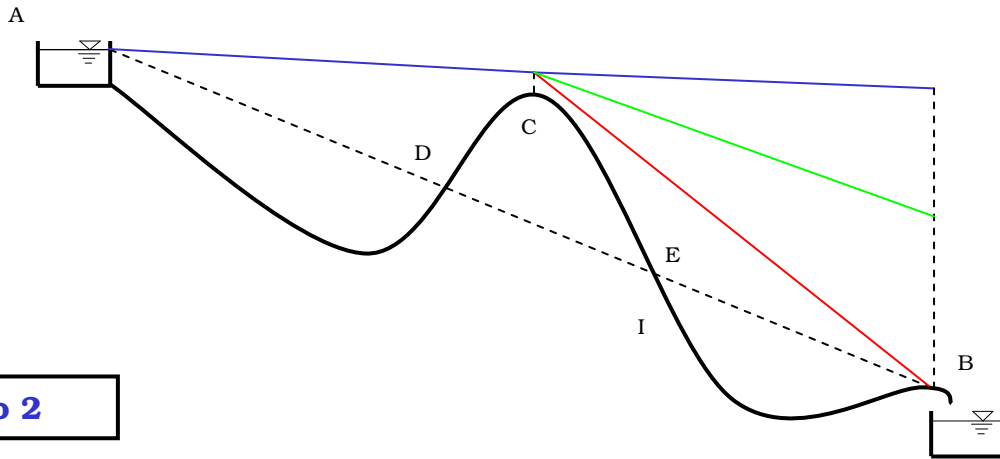


$$D_{\text{comm}} \geq D$$

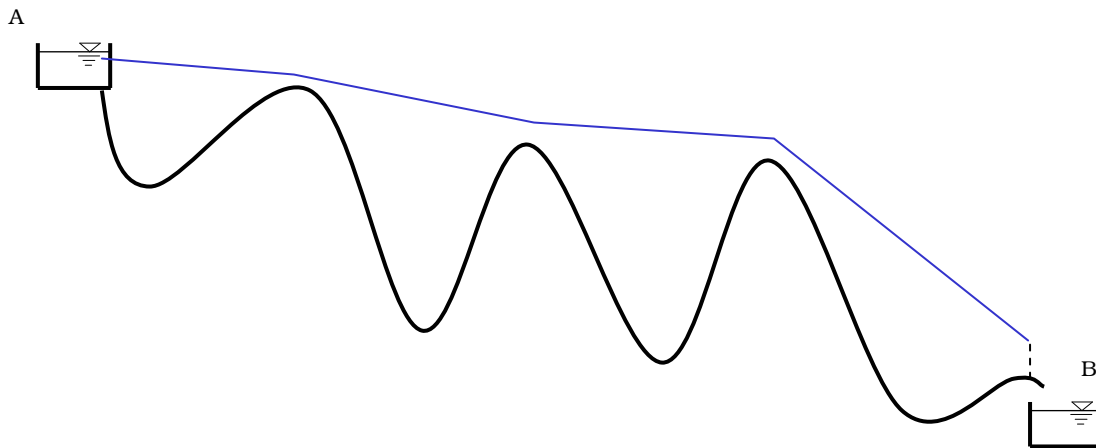
$$0.5 \leq V \leq 2 \text{ m/s}$$

$$\frac{P}{\gamma} > 5 \div 10 \text{ m}$$

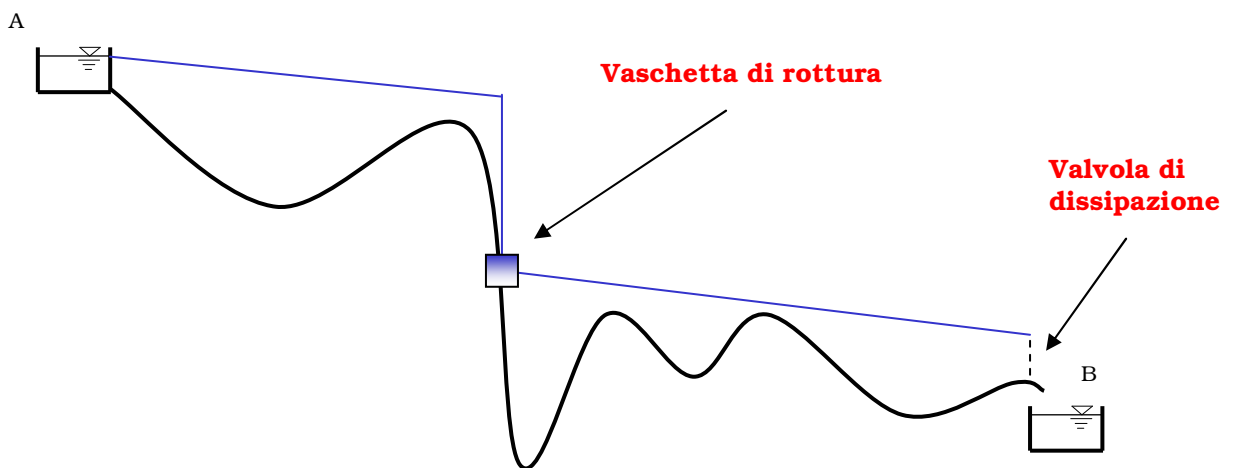
Caso 1



Caso 2

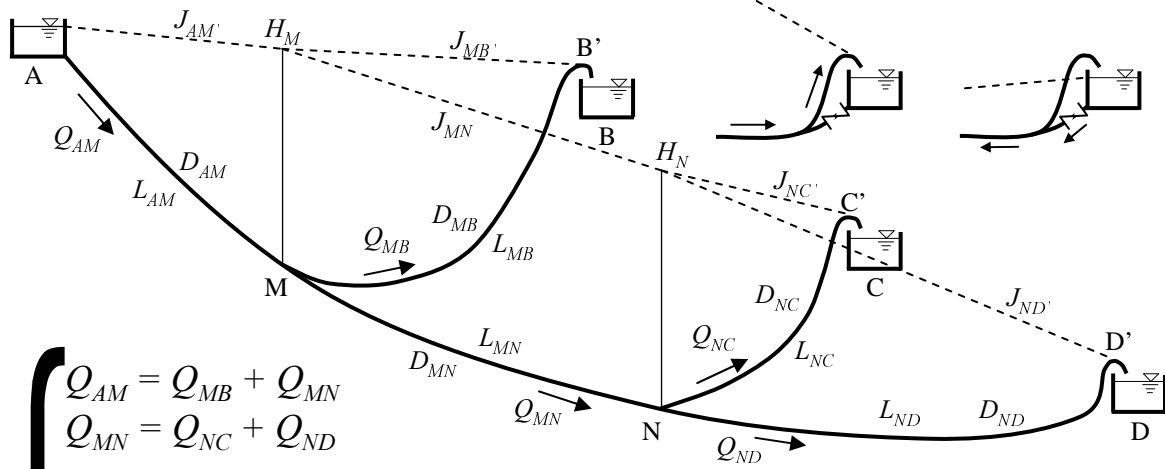


Caso 3



Sistemi di condotte a rami aperti

1) Problemi di verifica

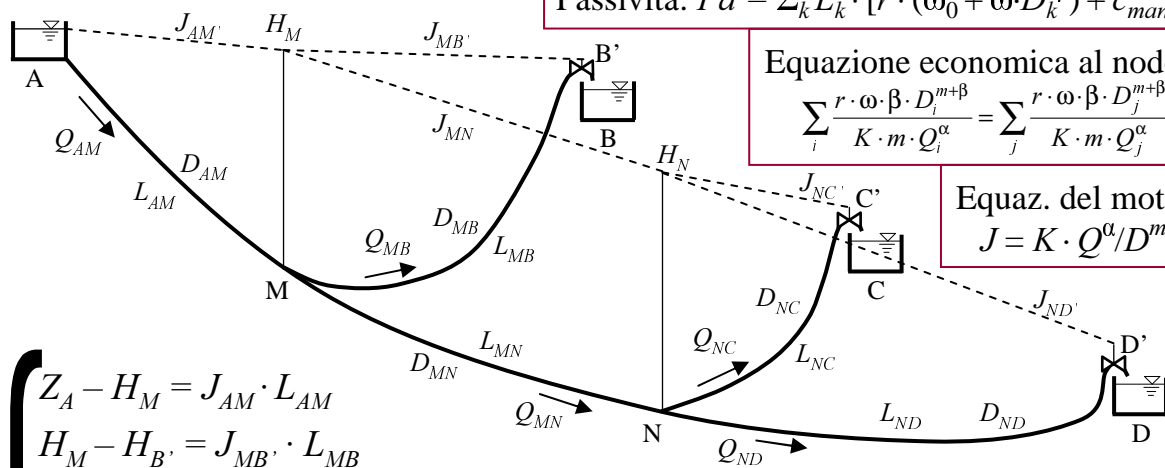


$$\left\{ \begin{array}{l} Q_{AM} = Q_{MB} + Q_{MN} \\ Q_{MN} = Q_{NC} + Q_{ND} \\ Z_A - H_M = J_{AM} \cdot L_{AM} \\ H_M - Z_{B'} = J_{MB'} \cdot L_{MB} \\ H_M - H_N = J_{MN'} \cdot L_{MN} \\ H_N - Z_{C'} = J_{NC'} \cdot L_{NC} \\ H_N - Z_{D'} = J_{ND'} \cdot L_{ND} \end{array} \right.$$

n carichi H incogniti sugli n nodi
 l portate Q incognite negli l lati

n equazioni di continuità per gli n nodi
 l equazioni del moto per gli l lati

2) Problemi di progetto



Costo unitario di costruzione: $c_i = \omega_0 + \omega \cdot D^\beta$

Passività: $Pa = \sum_k L_k \cdot [r \cdot (\omega_0 + \omega \cdot D_k^\beta) + c_{manut}]$

Equazione economica al nodo:

$$\sum_i \frac{r \cdot \omega \cdot \beta \cdot D_i^{m+\beta}}{K \cdot m \cdot Q_i^\alpha} = \sum_j \frac{r \cdot \omega \cdot \beta \cdot D_j^{m+\beta}}{K \cdot m \cdot Q_j^\alpha}$$

Equaz. del moto:

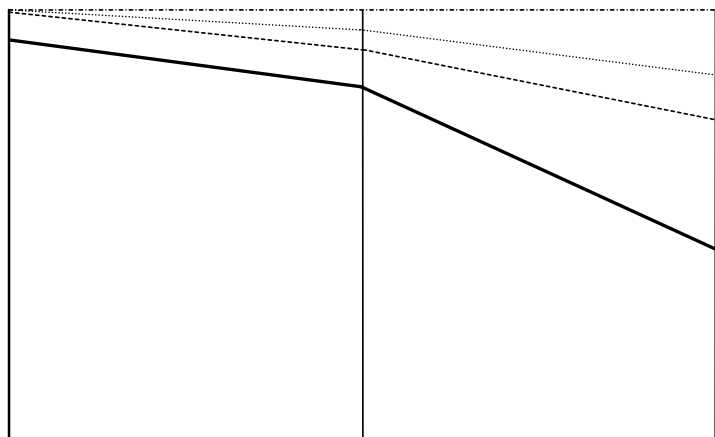
$$J = K \cdot Q^\alpha / D^m$$

$$\left\{ \begin{array}{l} Z_A - H_M = J_{AM} \cdot L_{AM} \\ H_M - H_{B'} = J_{MB'} \cdot L_{MB} \\ H_M - H_N = J_{MN'} \cdot L_{MN} \\ H_N - H_{C'} = J_{NC'} \cdot L_{NC} \\ H_N - H_{D'} = J_{ND'} \cdot L_{ND} \\ D_{AM}^{m+\beta} / Q_{AM}^\alpha = D_{MB}^{m+\beta} / Q_{MB}^\alpha + D_{MN}^{m+\beta} / Q_{MN}^\alpha \\ D_{MN}^{m+\beta} / Q_{MN}^\alpha = D_{ND}^{m+\beta} / Q_{ND}^\alpha + D_{NC}^{m+\beta} / Q_{NC}^\alpha \end{array} \right.$$

n carichi H incogniti sugli n nodi
 l diametri D incogniti negli l lati

n equaz. economiche negli n nodi
 l equaz. del moto per gli l lati

——— Asse condotta
 - - - - Piezometrica tubi usati
 Piezometrica tubi nuovi
 - . - - - Piezometrica statica



SEZIONI		①	②	③	
QUOTE [m s.l.m.]	Terreno	449.2	447.2		
	Asse condotta	448.0	446.0		
	Piezometrica tubi nuovi	451.0	448.0		
	Piezometrica tubi usati	451.0	447.5		
	Piezometrica statica	451.0	451.0		
DISTANZE [m]	Parziali	120	135		
	Progressive	0	120	255	
	Manufatti speciali	Parziali	350		
		Progressive	0		
DIAMETRI		φ 300			
MATERIALI					